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# An Investigation on Diffuser Augmented Wind Turbine Design



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A thesis submitted in partial fulfilment of the requirements for the degree of  
Doctor of Philosophy in Engineering

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The University of Auckland 2003

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Always there, their unfailing support and encouragement too often goes without the  
acknowledgement it deserves.

Sarah, Ma, Pa and Sis; it is with you I share this achievement and  
to you I dedicate this work.

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## *Abstract*

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Diffuser Augmented Wind Turbines (DAWTs) are one of many concepts to have been proposed to reduce the cost of renewable energy. As the most commercially viable, they have been the focus of numerous theoretical, computational, and experimental investigations. Although intimated in these studies to be able to augment the power output of a wind turbine, the extent of this power increase, or augmentation, the factors influencing DAWT performance, the optimal geometric form and their economical benefit remained unanswered. It is these issues that have been addressed in this investigation.

In reviewing historic investigations on DAWTs it has been identified that excessive wind tunnel blockage, inappropriate measurement technique, varied definitions of augmentation, and the inclusion of predicted performance based on incorrect assumptions have in general led to the overstatement of DAWT performance in those studies. In reassessing the performance of the most advanced of those DAWT designs, Grumman's DAWT 45, it has been calculated that the actual performance figures for the 2.62 exit-area-ratio and 0.488 length-to-diameter ratio DAWT were an available augmentation of 2.02, a shaft augmentation of 0.64 and a diffuser efficiency of 56%.

By contrast, the development of the Mo multi-slotted DAWT in this investigation has yielded a design whose shaft augmentation of 1.38 was achieved by a diffuser with exit-area-ratio of only 2.22 and overall length-to-diameter ratio of 0.35.

Such performance improvement has been obtained by gaining both an understanding of the flow characteristics of DAWTs and the geometric influences. More specifically it has been shown that: the velocity across the blade-plane is greater than the free-stream velocity and increases towards the rotor periphery; that the rotor thrust or disc loading impacts upon diffuser performance by altering the flow behaviour through it; and that DAWTs are able to maintain an exit pressure coefficient more negative than that attainable by a conventional bare turbine. The net result is that DAWTs encourage a

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greater overall mass-flow as well as extract more energy per unit of mass-flow passing through the blade-plane than a conventional bare turbine.

The major drivers of DAWT performance have been shown to be the ability of the design to maximise diffuser efficiency and produce the most sub-atmospheric exit pressure possible. Parametric investigation of the various DAWT geometric components has shown peak performance to be obtained when: the external flow is directed radially outward by maximising the included angle of the external surface in conjunction with a radially orientated exit flap; by applying boundary-layer control to a trumpet shaped diffuser via a pressurised cavity within the double-skin design of the multi-slotted DAWT; having an exit-area-ratio of the order of 2.22; and by employing an inlet contraction with inlet-area-ratio matched to the mass-flow passing through the DAWT under peak operating conditions.

To translate the available augmentation into shaft power a modified blade element method has been developed using an empirically-derived axial velocity equation. The resulting blade designs whose efficiencies reached 77%, twice those of Grumman, highlight the accuracy of the modified blade element method in calculating the flow conditions at the blade-plane of the multi-slotted DAWT. It was also noted that the rotor efficiencies remain below 'best practice' and therefore offer the potential for further increases in shaft augmentation. However, in order to achieve such gains, a number of limitations present in the current method must be addressed.

In assessing the likely commercial suitability of the multi-slotted DAWT a number of real-world influences have been examined. Shown to have little if any effect on DAWT performance were Reynolds number, ground proximity and wind shear. Turbulence in the onset flow on the other hand had the beneficial effect of reducing separation within the diffuser. Finally, DAWT performance was assessed under yaw misalignment where it was shown that the multi-slotted DAWT performed favourably in comparison to that associated with a conventional bare turbine.

The major drawback identified in the DAWT concept by this investigation was its drag loading and the fact that drag and augmentation were interdependent. The result is that the cost of a conventional DAWT is dictated by the necessity to withstand an extreme wind event despite the fact that augmentation is only required up to the rated wind speed. The overall conclusion drawn was that in order to optimise a DAWT design economically, and therefore make the DAWT concept a commercial reality, a creative solution that minimises drag under an extreme wind event would be required.

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## *Acknowledgements*

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Throughout such a course of study there are invariably numerous people without whom this accomplishment would not have been possible. Although acknowledged here, words alone do not do justice to the value of their support and assistance.

Firstly, I would like to thank my supervisors, Professor Richard Flay and Associate Professor Peter Richards. Their guidance, support and insight have been pivotal in the achievement of this degree. Their skilful use of sabbaticals has, I'm sure, enabled their enthusiasm to endure. Both academically and personally, the many discussions with Richard and lessons on and off the court from Peter have been invaluable. Thanks must also go to Professor Gordon Mallinson for his technical guidance, advice and hospitality during the development of computational models particularly during those dark Auckland days. Appreciation is also extended to the Mechanical Engineering staff who have provided much humour and competition over the years.

The impetus of this research was derived from the team at Vortec Energy. In particular, a special thanks is extended to Trevor Nash. Trevor's support, advice and assistance both professionally and personally have been invaluable.

The financial burden of postgraduate study has been eased with the support of the Maurice Paykel Graduate Scholarship and Vortec Energy. I would also like to acknowledge the Danish Centre for Applied Mathematics and Mechanics International Graduate Research Scholarship and in particular, Professor Martin Hansen, from whom I gleaned many valuable tools of the trade.

My final acknowledgement is extended to the many postgraduates I have meet along the way. Thanks to all those past and present who have distracted, been taught and have taught me many lessons throughout our battles together. Chris, Dan, Dave, Fai, Keith, Mat, Mike, Miro, Rhys, Sharlene and Simon have all shared this time with me.

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## Nomenclature

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### English Symbols

$A$	Area
$a$	Axial flow interference factor
$a'$	Tangential induction factor
$B$	Number of blades
$c$	Blade chord
$C_L$	Lift coefficient
$C_D$	Drag coefficient
	Disc loading coefficient relative to the free-stream dynamic pressure
$C_{t\infty}$	$\frac{\Delta P_{2-3}}{q_\infty}$
	Disc loading coefficient relative to the local dynamic pressure
$C_{t2}$	$\frac{\Delta P_{2-3}}{q_2}$
	Available power coefficient
$C_{p_{air}}$	$\frac{\Delta P_{2-3} A_r V_2}{\frac{1}{2} \rho V_\infty^3 A_r}$
	Shaft power coefficient
$C_{p_{sh}}$	$\frac{T \omega}{\frac{1}{2} \rho V_\infty^3 A_r}$
	Exit pressure coefficient
$C_{p_4}$	$\frac{\Delta P_{\infty-4}}{q_\infty}$
$D$	Drag force
$F$	Force
$g$	Acceleration due to gravity
$H$	Energy

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$k$	Screen loss coefficient
$L$	Lift force
$\dot{m}$	Mass-flow rate
$P$	Power
$p$	Pressure
$Q$	Volume flow rate
$q$	Dynamic pressure
$R_{tip}$	Tip radius of the rotor
$r$	Local radius of the rotor
$r_{air}$	Available augmentation $\frac{C_{p_{air}}}{0.593}$
$r_{sh}$	Shaft augmentation $\frac{C_{p_{sh}}}{0.593}$
$Re$	Reynolds number
$T$	Torque
$V$	Velocity
$W$	Resultant velocity
$x$	Distance in axial direction from DAWT inlet
$z$	Height above ground

**Subscripts**

cb	Quantities evaluated at/related to the centre body
diff	Quantities evaluated at/related to the diffuser
r	Quantities evaluated at rotor blade-plane/related to the rotor
$\infty$	Station in ambient free-stream
1	Station at DAWT inlet
2	Station immediately upstream of blade-plane
3	Station immediately downstream of blade-plane
4	Station at diffuser exit-plane
5	Station well downstream of turbine



**Greek Symbols**

$\alpha$	Angle of attack
$\phi$	Resultant flow angle
$\theta$	Local pitch angle
$\varepsilon$	Velocity speed-up $\frac{V_2}{V_\infty}$
$\eta$	Efficiency
$\lambda$	Reciprocal of the diffuser exit-area-ratio $\frac{1}{\left(\frac{A_4}{A_3}\right)}$
$\mu$	Dynamic viscosity
$\sigma$	Rotor solidity
$\nu$	Kinematic viscosity
$\rho$	Density
$\omega$	Rotational speed of rotor

**Abbreviations and Acronyms**

DAWT	Diffuser Augmented Wind Turbine
EAR	Exit-area-ratio $\frac{A_4}{A_3}$
HAWT	Horizontal-Axis Wind Turbine
TSR	Tip-speed ratio

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